18TH WORLD ENERGY CONGRESS

HEAT PUMPS FOR DIFFERENT WORLD REGIONS – NOW AND IN THE FUTURE

POMPES DE CHALEUR POUR DIFFERENTES PARTIES DU MONDE – AUJOURD'HUI ET A L'AVENIR

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1. INTRODUCTION INTRODUCTION

Heat pumps offer the possibility of reducing energy consumption significantly, mainly in the building sector, but also in industry. The second law of thermodynamics shows the advantages: While a condensing boiler can reach a primary energy ratio (PER) of at best 105 % (i.e. the boiler efficiency η_B ; the theoretical maximum would be 110 %, based on the lower calorific value), heat pumps achieve 200 % and more.

Whereas the thermodynamic principle of the heat pumping process was found at the beginning of the 19th century (by Carnot, Kelvin, and others), the heat pumping process was realized about 1834 for refrigeration, and not before 1855 for producing heat: In this year, Peter Ritter von Rittinger put into operation the first heat pump, an open-cycle mechanical vapour recompression (MVR) unit, directly driven by hydro energy, for producing salt from brine in the saltern of Ebensee, Upper Austria. Much later, also the closed vapour process, as used for refrigeration, consisting of evaporation, compression, condensation, and throttling, was used for generating useful heat. Essentially after World War II, the small heat pumping units for air conditioning of homes and individual rooms became common, somewhat later the "reversible" units for coooling/dehumidifying as well as heating, and after the oil price crisis of 1973 also the heating-only heat pumps for heating in moderate and cold climates.

The refrigerants (working fluids) up to the 1930s were ammonia, carbon dioxide and other fluids, most commonly toxic or/and flammable; later on the "safety refrigerants" (chlorofluorocarbons, CFCs, and chlorofluorohydrocarbons, HCFCs) quickly occupied the market. This remained so until at the Vienna Convention of 1986 and the Agreement of Montreal 1997, the future production of CFCs was limited and in the end essentially banned, for reasons of destroying the stratospheric ozone layer. A few years later (Kopenhagen, 1992) also an agreement for limiting the HCFCs was concluded. The first choice of replacement fluids were (chlorine-free) hydrofluorocarbons, and their mixtures. However, whereas these have no ODP (ozon depletion potential), they have a GWP (global warming potential) when liberated during operation of the heat pumping unit or at the end of its life. There is, therefore, a tendency in some, mostly European, countries to switch again to "natural" working fluids, containing neither chlorine nor fluorine and a negligible GWP. Already in use are propane and iso-butane (which are flammable), NH_3 (which is poisonous), CO_2 , and (for special applications) water.

So the heat pump has undergone and is undergoing several changes. The changes in working fluids have entailed a number of design changes. However, the efficiency today is generally better than before the changes and keeps rising. Thus, not only the environmental effects of the working fluids are being reduced, but also the effects of power plants producing the driving energy for the heat pumps – due to higher SPFs, higher power plant efficiencies (η_{PP}), and an energy sources mix with lower mean GWP.

2. BASICS BASES

2.1. Definitions Définitions

The general term heat pumping technologies is used for processes in which the natural heat flow from a higher to a lower temperature level is reversed by adding high value energy, i.e. exergy. The term heat pump is used in a different way in different regions of the world. In Japan and in the USA reversible units are called heat pumps: they offer, besides cooling and dehumidification, the capability of providing useful heat. Most commonly air to air units are used; however, ground-coupled systems have an increasing share. Chillers are more or less always called chillers, even if they are as heat pump chillers also used for heating purposes. In Europe the term heat pump is used for heating-only units with the heat sources outside air, ground and ground water, combined with hydronic heat distribution systems.

For industrial applications the term heat pump is often replaced by other names like Mechanical Vapour Recompression (MVR) System, dryer, dehumidifier etc., and in heat pump based heat recovery systems the term heat pump is very often not even mentioned.

2.2. Efficiencies Rendements

<u>Figure I</u> demonstrates the efficiencies of thermodynamic heating/cooling. Over the (positive or negative) effective temperature lift ΔT from ambient, the relative exergy E_X/Q is plotted for the ideal process (Carnot process, second law of thermodynamics) and for practical real processes (hatched areas).



- Figure I Ideal and real power consumption E_x/Q for cooling (freezing, refrigeration, air conditioning) and space heating by heat pump and by cogeneration district heating
- Figure I Consommation de puissance idéal et réel pour réfrigération (surgeler, réfrigérer, climatiseur) et pour installation de chauffage avec pompe à chaleur et avec chauffage urbain cogeneration

For the ideal process:

$$E_x/Q = 1 - T_a/T = (T - T_a)/T = \Delta T/T = \eta_c$$

where

Ex Exergy

- Q Heat transferred
- T_a Ambient temperature, K
- T Process temperature, K
- η_c Ideal (Carnot) efficiency

The coefficient of performance (COP) is shown at the right-hand scale: COP = Q/E_x . The internal efficiency is given by the ratio $\eta = COP/COP_{ideal}$ at ΔT .

The left-hand area refers to cooling: freezing, refrigeration and space cooling (air conditioning including dehumidification). For these applications, E_x/Q ranges from 0.1 to 0.5, the COP therefore from 2 to 10.

The right-hand area refers to heating: The heat pump area shows a temperature lift of 5 to 70 K, E_x/Q is between 0.08 and 0.45 and COP therefore between 2.2 and 12.5, the higher value referring to the opencycle industrial heat pump (Mechanical Vapour Recompression, MVR). For heat pumping technologies the efficiency η is about 0.4...0.7. The area "Cogeneration" is different. Here the real exergy loss is smaller than the theoretical one: The actual loss of electricity due to extracting steam for district heating is smaller than the exergy of the extracted steam, because of turbine and condenser losses. Despite this, COP is not or not much larger than for the heat pump: District heating requires higher temperature lifts, and there are piping losses (which usually are larger than electricity distribution losses).

2.3. Performance Factors Coefficientes de rendement

The drive energy of heat pumps is most commonly electricity, and for the future improved power generation systems based on renewables and fossil fuels have to be taken into consideration. The power plant efficiency, η_{PP} , is up to 58 % for gas-fired combined-cycle power plants available on the market, with oil as fuel similar values are possible. Ground-source ("geothermal") heat pumps combined with low-temperature heat distribution systems achieve seasonal performance factors (SPFs, ratios of useful heat to electricity as drive energy) of 4 and higher, which means primary energy ratios PER = SPF. η_{PP} , (ratios of useful heat to primary energy) of 220 to 280 %. However, improvements are possible in the future.

The power plant efficiency η_{PP} depends, of course, on the kind of fuel (primary energy source). <u>Table I</u> shows the relations for the more important primary energy sources and for SPF = 4 and 5. PER is highest for direct power generation from renewable sources such as hydro or wind, for which $\eta_{PP} = 1.0$ by definition. Sometimes PER is assessed for a national average η_{PP} , i.e. for the national power generation mix.

PER gives an absolute measure of the units of useful heat obtained from one unit of primary energy at power plant input, neglecting for the moment losses upstream of the power plant such as in production, cleaning, transmission (WEC, 1988), and distribution losses between power plant and heat pumps.

More information than by PER is, however, given by comparing, for a given fuel (primary energy source), the efficiency of the indirect path via power plant and heat pump (PER) to the efficiency of the direct path of conversion (η_B), e.g. in a heating boiler. The ratio may be called Useful Energy Ratio UER = PER/ η_B . It is also given in Table I. Comparing the same fuel means that all upstream effects cancel each other out. The downstream effects, i.e. the local distribution losses, also cancel each other out in the case of electric heating from hydro, wind, or nuclear plants whereas they may be considered of equal value as a first approximation when comparing electricity for the heat pump, and fuel for the boiler. If, in the latter case, the distribution efficiencies η_d should be markedly different, a more exact formulation would be:

 $\mathsf{UER} = (\mathsf{PER}/\eta_\mathsf{B})(\eta_{\mathsf{d},\mathsf{ee}}\!/\eta_{\mathsf{d},\mathsf{fuel}})$

where $\eta_{d,el}$ relates to electricity, and $\eta_{d,fuel}$ to the fuel distribution efficiency.

Table I Typical Primary and Useful Energy Ratios

Table I Proportions typiques de l ènergie primaire et utile

	Coal (and Biomass)	Gas	Electricity from Renewables (hydro, wind)	Nuclear
<u>Efficiencies</u> Power plant, η _{ΡΡ} Boiler (local conversion), η _Β	0.4 0.8	0.55 0.98	1.0 1.0	0.33 1.0
$\frac{\text{PER for SPF} = 4}{\text{PER} = \text{SPF. } \eta_{\text{PP}}}$ $\text{UER} = \text{PER}/\eta_{\text{B}}$	1.6	2.20	4.0	1.33
	2.0	2.24	4.0	1.33
$\frac{\text{PER for SPF} = 5}{\text{PER} = \text{SPF. } \eta_{\text{PP}}}$ $\text{UER} = \text{PER}/\eta_{\text{B}}$	2.0	2.75	5.0	1.67
	2.5	2.81	5.0	1.67

The data of Table I show that

- UER is larger or − for η_B = 1.0, as for electric heating − equal to PER. Only for η_B somewhat higher than 1.0 − as may be the case for very efficient gas-fired condensing boilers − or for η_{d,el} < η_{d,fuel} could UER become lower than PER.
- For direct electricity from renewables, the efficiencies are 1.0, and UER = PER = SPF.
- For the basic data of Table I, UER ranges from 1.33 to 5.0.
- Boiler efficiencies near 1.0 are close to the theoretical limit (i.e. for the gas-fired condensing boiler). A SPFs of around 5.0 is far below the theoretical limit of heat pumps (which is about 14 at 0°C ambient temperature); SPFs of 6 or more may be possible and will be economic in the future.

For absorption heat pumps, PER is the ratio of heat output to primary energy input (not to the power plant but to the heat pump).

2.4. Renewable energy gain by heat pumps Gain d` énergie renouvelable par pompes à chaleur

It should be noted that the heat pump, which in most cases grades up free heat from the environment (air, water, ground) and from waste heat, is a major source of renewable energy. The renewable heat R gained by the heat pump is the difference between the thermal output Q and the drive energy E_x (in the case of electricity, $E = E_x$):

R = Q - E = Q - Q/SPF = Q(1 - 1/SPF)

Obviously, if the drive energy is electricity from renewable sources, all the energy used for the heat pump is renewable energy.

2.5. Combining CHP and heat pumps Cogeneration combiné avec pompes à chaleur

It should also be noted that the combination of electric heat pump and combined heat and power (CHP) in coal, gas or nuclear stations yields a somewhat higher PER and UER (Frutschi, 1991). E.g. if in a coal power station, η_{PP} ` = 0.35 and the heat efficiency η_H = 0.4, then for SPF = 4, PER` = (4.0) (0.35) + 0.4 = 1.4 + 0.4 = 1.8, and UER` = PER`/ η_B = 2.25 rather than 2.0 as per Table I.

3. APPLICATIONS APPLICATIONS

The sectors where heat pumps are used can be divided in the building sector (residential and commercial), where the majority of heat pumps is in operation, and in the industrial sector.

3.1. Residential/Commercial Secteur des batiments et des commerces

The majority of heat pumps in operation in the residential sector are reversible air-to-air air conditioners for both heating and cooling. The types most commonly used in the small to medium capacity range are

1) Window-type

They are mainly popular in the USA, Brazil, Australia, Saudi Arabia, the Philippines, India, Thailand and Hong Kong.

2) Split-type room air-conditioner (Cooling capacity under 5 kW)

They are popular in Japan and China, and also becoming the predominant type in South Korea, Thailand and Malaysia.

The medium-sized packaged air-conditioners (PAC) suited for office use are classified in the following two types:

- The unit packaged type air conditioner (unitary air conditioner) which are manufactured and sold mainly in USA. They are also becoming the major types in Australia, the Middle East, Canada and Mexico.
- 2) The split-type and multi-split-type packaged air conditioner (cooling capacity over 4 kW) are popular mainly in Japan, South Korea and China. Split-type and especially multi-split-type units have been improved significantly as the market expands, including the development of inverter-control methods which regulate air-conditioning performance by varying compressor speed according to thermal load; and simultaneous cooling and heating functions in the multi-split-type air conditioner.

These reversible air-to-air units are very cost-effective due to the fact that the additional feature of dualmode operation is relatively cheap and due to the long annual operation time.

Europe has been concentrating on heating-only units with ground water, the ground or outside air as heat source, integrated into hydronic heat distribution systems. Ground-coupled systems combined with low-temperature heat distribution systems achieve seasonal performance factors of 4 and higher. However, the market share of air-to-air split air conditioners is rapidly increasing.

In larger systems, especially in commercial buildings, chillers and heat pump chillers are in operation, sometimes water-loop heat pumps are used instead of four-pipe distribution systems with fan-coil units. In Asia sorption - both absorption and adsorption - systems become popular, the main purpose of these units is load levelling of electric power requirement. Drive energy is most commonly gas, but more and

more systems driven by heat from co-generation plants are in operation. The alternative are electrically driven units with ice-storage systems.

Two other types in operation in the residential sector are heat pump water heaters and large units for district heating systems. Large heat pumps with two-stage centrifugal compressors for district heating supply have been built and are operated mainly in Sweden. The heat capacities of these units are usually between 10 and 14 MW per unit, but unit capacities go up to 45 MW. The drive energy is electricity, heat sources are treated sewage water, sea water or industrial waste heat.

3.2. Industry Secteur d`industrie

The industrial heat pump (IHP) in the strict sense is a unit operation, usually heat recovery, within an industrial process, (IEA HPC, 1997a, 1997b; Stuij, 1994) mostly in the chemical, food and lumber industries. The heat pump may be seen as a heat exchanger, working on a negative temperature difference, i.e. the temperature of the heat receiving flow in higher than the one of the heat supply flow. Then, the heat pumps yields more freedom in designing a process integration chain.

The heat pump may be of the closed-cycle type using a separate working medium as in other energy sectors, or of the open-cycle type (mechanical vapour recompression, MVR, as in the first heat pump mentioned in chapter 1), where the process medium is also the working medium of the heat pump; one heat transfer loss can be omitted, which increases the efficiency of these systems.

Absorption.units are used if cheap process heat is available as drive energy. The heat transformer if used if there is excess heat available at a medium temperature level and high temperature heat is required.

Whereas in the residential sector there is typically a large number of small units, in industry there is a relatively small number of large heat pumps.

In many cases, waste heat from industrial processes is upgraded by a heat pump to be used for space or water heating inside or outside a factory. There is also air conditioning for the process and/or for humans. Often these applications are counted as industrial heat pumps.

4. PRESENT SITUATION SITUATION PRÉSENTE

4.1. Definition of Major Regions Définition des Regions Grandes

The application of heat pumps strongly depends on the climatic conditions and on the building standards.

- ♦ The main market are regions with moderate winters not below -5°C and summers which require cooling and dehumidification. This is the area of air/air heat pump air conditioners, which are prefabricated, which can be easily installed and which, due to a long annual operation time, are highly cost-effective.
- In regions with cold winters, where additionally cooling is required in summer, the heat pump has to compete with conventional heating systems. Air as heat source for heating purposes is due to low SPFs not the best solution in this case.
- Regions with cold winters and no real need for cooling in summer can be a market for heat pumps if heat sources like ground water or the ground can be used. Heat from the ground can be extracted either by horizontally installed collectors (cost-effective in the case of a new building, if enough area is available) or through bore holes (more expensive).

The world regions were defined as follows: The majority of heat pumps is in operation in Japan, China and in the US, most commonly air-to-air dual-mode (reversible) units for both heating and cooling. In Europe, heating-only heat pumps are used mainly in the Northern and in the Central part, reversible units in the Southern part. Remaining countries are grouped under "Others".

4.2. Markets in Regions Marchés en Régions

4.2.1. Japan

Japon

The number of heat pumps installed in Japan was 39.3 million in 1992 (Stuij, 1994), and 58 million units (68 % of the world total of 85 million units) in January 1997 (Breembroek, 1999). In the year 2000, the number may have reached about 67 million.

The number of room air conditioners (RAC) shipped in 1996 has reached a peak of abourt 8 million in 1996 of which 92 % were equipped for heat pump operation; in 1997 the figures were more than 7 million and 94 %, respectively (Figure II). The diffusion rate of heat pumps was 62 %, up from 35 % in 1987; it is gradually approaching saturation (Nishimura, 1999; Dinghuan, 1999). The annual production of packaged air conditioners, mainly for commercial/office buildings, is around 800,000 (Dinghuan, 1999).



Figure II Domestic shipment of Room Air Conditioners in Japan (Dinghuan, 1999) Figure II: Nombres de vente du pays des conditionneurs d'air individuelles en Japon (Dinghuan, 1999)

An increasing number of larger heat pump systems are equipped with thermal storage systems <u>Figure III</u>): water storage, mainly underground, up to 1993, mainly ice storage after that date. The shipment of engine-driven heat pumps in Japan has reached more than 40,000 units in 1997 <u>Figure IV</u>), 95 % of them utilising gas engines (GHP), the remainder kerosene (KHP).

Also the number of absorption systems is increasing, although mainly for cooling. <u>Figure V</u> shows the cumulative cooling capacity in 10^6 RT (Refrigeration tons, where 1 RT = 3.5 kW), 6 million RT therefore being 21 GW.



Figure III Thermal storage type heat pump systems in Japan (Nishimura, 1999) Figure III Nombres des stockages thermiques des pompes à chaleur en Japon (Nishimura, 1999)



Figure IV Domestic shipment of engine-driven heat pumps in Japan (Dinghuan, 1999) Figure IV Nombres de vente du pay des pompes à chaleur êntrainé par moteur en Japon (Dinghuan, 1999)



Figure V Absorption Systems and GHPs in Japan (Nishimura, 1999) Figure V Pompes à chaleur d'absorption et entrainé par moteurs au gaz en Japon (Nishimura, 1999)

4.2.2. China La Chine

Production of RACs in China was low until 1990, when a rapid upswing started, bringing the production to about 5 million units in 1997, probably around 7 million in 2000, and more in 2001 (Figure VI). This means that China has already the largest industry world-wide for producing air conditioners. The heat pump share was 60 % in 1997, and the number of heat pumps installed was 11.4 million in 1997, slightly more than in the US (Breembroek, 1999). Presently, the heat pump stock may be of the order of 25 million. Considering this rapid development, the IEA HPP has accepted the offer of the Chinese government to be the host of the 7^{th} IEA Heat Pump Conference "Heat Pumps – Better by Nature" in Beijing in May 2002.



Figure VI Room Air Conditioners installed in China (Nishimura, 1999) Figure VI Nombres des conditionneurs d'air individuelles installè à Chine (Nishimura, 1999)

4.2.3. USA États Uni

The number of heat pumps installed in the USA was 9.5 million in 1992 (Stuij, 1994) and 11.1 million in 1997 (Breembroek, 1999). Annual sales are about 1.2 million, partly for replacement. The present number may therefore be 13 or 14 million. Numbers include annual sales of 60.000 ground-coupled (geothermal) heat pumps.

The problem in the USA are the climatic conditions. Almost in the whole country air conditioning is required, but air-to-air units cannot cover the winter peaks with extremely low temperatures. This results in systems where air conditioners are used for cooling and gas furnaces for heating. The situation in Canada is very similar, considering the populated regions of this country.

4.2.4. Europe Europe

After the oil crisis Europe has been concentrating on heating-only hydronic heat pumps and heat recovery systems, but nowadays sales of dual-mode units are growing.

The number of heat pumps in Europe is given as 4.3 million units in 1997 Breembroek, 1999; Bouma, 1999). The majority of these are reversible air-to-air systems in the residential and commercial sectors in Southern Europe, mainly imported from USA and Japan. The share of these units is increasing. The number of heating-only units and heat recovery units is in the range of 1.5 million units.

A real heat pump market exists in Sweden, other countries have started heat pump programmes -Switzerland, Germany, and the Netherlands – or they are just preparing programmes to increase the market break-through of this technology. The deregulation of the electricity market promises lower tariffs for electricity, which could be an advantage for heat pumps; however, the activities of many electric utilities seem to be at the moment concentrated on re-organisational matters caused by the deregulation and liberalisation of the market and not so much on heat pumps.

4.2.5. Others Autres

In the 1993-1996 market review of the IEA Heat Pump Centre (Breembroek, 1997), there were only 0.6 million units listed for regions other than the four mentioned above (0.5 million for Canada, and less than 0.1 million for South Korea). So the total of

Japan	57.6 m	6 million		
China	11.4	"		
USA	11.1	"		
Europe	4.3	"		
Others	0.6	"		
	85.0 m	0 million		

was obtained for 1996/beginning of 1997. But there are other markets (Dinghuan, 1999) such as

- Hong Kong,
- South East Asia (Indonesia, Thailand, Malaysia, Singapore, Philippines),
- India,
- Australia/New Zealand

with RAC/PAC annual scales of about 3 million units per year, from which a total number of installed units of about 25 million may be estimated. However, these are mainly cooling-only units and not heat pumps, except the ones in Australia/New Zealand (in total perhaps 3 million units). Adding another estimated 7 million units for South America, Mexico, South Africa, Middle East and the remaining countries, the global number for 1996/97 may be 95 million rather than the above 85 million units for 1997.

The world total number of the RAC and PAC market (cooling only as well as reversible) is estimated to be of the order of 33 million units sold per year (Nishimura, 1999).

The split-up of the above 85 million units quoted by the IEA HPC for 1997 is 75 million residential, almost 10 million commercial and about 20,000 industrial units.

4.2.6. Total

Nombre total

According to the HPC market study (Breembroek, 1999) the world-wide heat pump stock has increased in the four years 1992 to 1996 from 55 to 85 million units, i.e. by 55 %. Assuming for the four years from 1996/97 to 2000/01 a slightly lower increase (due to lower economic growth in Asia during that time), the global number in the year 2001, starting from the 95 million deducted above rather than the 85 million units, may be of the order of 130 to 140 million heat pumps in operation world-wide.

4.3. Thermal output Sortie thermique

The thermal output of the world's heat pumps in estimated for 2001 from the data of Table IV presented in the next chapter as follows:

- Residential: 75 million units, 10.000 kWh/a each: 750 TWh/yr
- Commercial 350 " - Industrial 200

-	inuusinai		200
		Total	1300 TWh/yr.

According to the relation given in chapter 2.3, the renewable energy R gained by the heat pump, i.e. the thermal input, for SPF = 3 is R = (1 - 1/SPF) Q = 0.67 Q. For Q = 1300 TWh/yr, as above, the renewable energy R presently gained by heat pumps becomes 870 TWh/yr.

4.4. Diffusion Diffusion

The diffusion of the heat pump, i.e. the share of useful heat produced by it, is negligible in the transport sector, low in industry, but already substantial in the residential/commercial sector of some countries, as is shown in <u>Figure VII</u>. In Japan, this share has reached 20 % for the building stock, in Sweden 9.4 %, in the US 6.9 %, in Spain 5 % (Laue, 1999). The share is increasing since for new building it is much higher. In Switzerland, more than one third of new homes is heated by a heat pump, so the present share of 1.7% will increase quickly.



Figure VII Diffusion of heat pump in the residential sector (heat demand met by heat pumps), % Figure VII Diffusion des pompes à chaleur au secteur rèsidentiel (besoin chaleur etableé par pompes à chaleur), %

5. POSSIBLE FUTURE DEVELOPMENT DEVELOPMENT POSSIBLE A L'AVENIR

5.1. Technological development Development technologique

An example for the development is the increase of COPs, e.g. for Room Air Conditioners from a cooling/heating COP of 2.2/3.1 for the 1982 model to 4.1/4.3 for the 1999 model (toh, 1995). Similar data (COP = 4.29) are given by Stuij (1995). One of the driving forces for improvements are labelling schemes.

In the small to medium size capacity range the reciprocating compressor has been practically replaced by the scroll compressor, and liquid/refrigerant heat exchangers have changed to welded flat plate heat exchangers.

Ground-coupled heating-only heat pumps, especially direct-evaporation systems in Europe, have increased their SPF to 4 and lately to 5 and more. Besides better components, improved building codes with the possibility of reducing supply temperatures required to values below 35°C are responsible for this development.

A similar development took place with chillers (Figure VIII).



Figure VIII Development of typical centrifugal chiller COP (Ertinger, 1999) Figure VIII Development de COP typique par turbo compresseur refriodisseur d'eau (Ertinger, 1999)

5.2. Market development Development du Marché

The possible future development can be assessed in two ways:

- by extrapolating the annual production as per Table II, part (1), and adding it to the total installation, part (2), accounting for numbers to be taken out of service, and by subsequently assessing the overall thermal output as per parts (3) and (4) of Table II (bottom-up method), or
- by taking the final energy or the three main sectors (industry, residential/commercial, transport) from statistics estimating a maximum share of each sector to be possibly and sensibly covered by heat pumps; also a (lower) share to be covered by some point in time, say the year 2020, which has to be related to the data of Table II, part 4 for the year 2000 (top-down method).

6. REDUCTION OF GLOBAL WARMING BY THE HEAT PUMP RÉDUCTION D' ÈCHAUFFEMENT DE TERRE AVEC POMPES À CHALEUR

According to the Kyoto Agreement, the global emission of greenhouse gases, in particular of industrialised countries, is supposed to be reduced. Of the six greenhouse gases mentioned in the Kyoto Agreement, CO_2 is the most important one (it is responsible for considerably more than 50% of the global warming effect) and at the same time it is the one the emissions of which are most difficult to be reduced worldwide. However, it can be shown that the heat pump is one of the key technologies for energy conservation and reducing CO_2 emission (IEA HPC, 1997).

<u>Table II</u> gives present and estimated future savings of CO_2 emissions due to the utilization of heat pumps in the residential and commercial sector as well as in industry.

The first column of Table IV is based on or derived from the data of IEA HPC (1977). It shows that in 1997 the heat pump saved already 0.5 % of the total global CO_2 emissions.

The second column is an extrapolation of the 1977 data to 2001 according to the data of chapter 4.

The saving potentials shown in the third and fourth column is again based on data of IEA HPC (1997). The third column refers to the potential savings (6 %) of CO_2 emissions by improved market penetration (30% in the building sector) using presently available technologies.

These 6% are one of the largest contributions to CO_2 reduction a single technology available on the market can offer. The fourth column is based on greatly advanced future technologies of heat pumps and power plant efficiencies. It yields a 16 % saving of global total CO_2 emissions.

Table II Present and estimated future savings of CO₂ emission by the use of heat pumps

Table II Économisations d'emissions de CO₂ par utilisation des pompes à chaleur, présentes et estimés par l'avenir

		1997 ¹	2001 ²	Savings Potential ¹	
				Present	Future
(a) <u>Residential</u> Annual heat demand per residence	kWh	10,000	10,000	9,000	8,000
Specific CO ₂ emissions from heat pump kg CO ₂ /kWh heat from oil-fired boiler "		0.215 ³ 0.713 ⁴	0.2 0.7	0.18 0.67	0.12 0.64
Number of residential HP CO ₂ emissions from oil-fired boilers	10 ⁶ MtCO ₂ /yr	65 204	70 215	670 1672	1,550 3,500
from heat pumps savings by HP	"	140 64	140 75	1022 650	1,500 2,000
(b) Savings Commercial	MtCO ₂ /yr	30	35	350	1,100
(c) Savings Residential + Commercial	MtCO ₂ /yr	94	110	1,000	3,100
(d) <u>Savings Industry</u>	MtCO ₂ /yr	20	22	200	600
(a) <u>Total Savings</u>	MtCO ₂ /yr	114	132	1,200	3,700
(f) Percentage CO_2 emission savings by heat pumps ⁵⁾		0.5%	0.6%	6.0%	16.0%

1. from IEA HPC (1997) or deducted from it

2. estimated

4. for 80 % efficiency in 1997, increasing to 90 %.

5. 1997 annual global CO₂ emission: 22 billion tonnes.

7. CONCLUSIONS CONCLUSIONS

Heat pumps are an old technology, which has not been extensively used as long as both energy prices and the efficiency of electricity generation have been low. The oil crises have changed this situation, and Kyoto is a further reason for the increasing market deployment of this technology. Based on recent developments, the following conclusions can be drawn:

Heat pumps offer the possibility of reducing energy consumption significantly, mainly in the building sector, but also in industry. Basic second law thermodynamics show the advantages: while a condensing boiler can reach a primary energy ratio (PER) of 105 % (the theoretical maximum would be 110 % based on the lower calorific value), heat pumps achieve 200 % and more, with hydro or wind energy even 400 % and more.

^{3. 0.215} kg CO₂/kWh heat (for 0.55 kg CO₂/kWh electric energy according to the European fuel mix and SPF = 2.5), decreasing to 0.12 for improved power plant efficiency and SPF and reduced fossil fuel in the mix.

- The majority of heat pumps is in operation in Japan, in China and in the US, most commonly air-to-air dual-mode units for both heating and cooling, China has already the largest industry for producing air conditioners, and in South East Asia the trend to this technology is rising rapidly.
- Europe has been concentrating on heating-only hydronic heat pumps and heat recovery systems, but sales of dual-mode units are growing.
- The drive energy is most commonly electricity, and for the future improved power generation systems based on renewables and fossil fuels have to be taken into consideration. The efficiency of gas-fired combined-cycle power plants available on the market is about 58 %, with oil as fuel similar values are possible. Ground-source ("geothermal") heat pumps combined with low-temperature heat distribution systems achieve seasonal performance factors (SPFs) of 4 and higher, which means PERs of 220 to 280 %. Further improvements are possible in the future.
- Sorption systems absorption, adsorption and DEC systems also gain importance, either for load levelling measures or together with CHP systems providing the drive energy. The efficiency of absorption units has been improved significantly by introducing welded flat plate heat exchangers for reducing heat transfer losses.
- Presently more than 100 million heat pumps with a thermal output of 1300 TWh/a are in operation world-wide, reducing CO₂ emission by about 0.13 Gt/a. The potential for reducing CO₂ emissions assuming a 30 % share in the building sector using technology presently available is about 6 % of the total world-wide CO₂ emission of 22 Gt/a. With future technologies up to 16 % seem possible. Therefore, heat pumps are one of the key technologies for energy conservation and reducing CO₂ emission.

REFERENCES RÈFERENCES

Bouma, J. (1999) Heat Pump Markets in Europe, Plenary Presentation 6, 6th IEA Heat Pump Conference "Heat Pumps – a Benefit for the Environment" (May 31 – June 2, 1999, Berlin

Breembroek, G., Lazaro, F. (1999), International heat pump status and policy review 1993-1996, Part 1 – Analysis. IEA Heat Pump Centre, Sittard, the Netherlands.

Breembroek, G. (1999a), Global environmental benefit of heat pumps for mitigating global warming. Poster Presentation 87, 6th IEA Heat Pump Conference "Heat Pumps – a Benefit for the Environment" (May 31 – June 2, 1999 Berlin).

Dinghuan, K., Iwatubo, T. (1999), Market in Asia Pacific. Plenary Presentation 5. 6th IEA Heat Pump Conference "Heat Pumps – a Benefit for the Environment" (May 31 – June 2, 1999, Berlin).

Ertinger, R., Brasz, J. (1999), Chillers - Entering the Next Century. Plenary Presentation 28, 6th IEA Heat Pump Conference "Heat Pumps – a Benefit for the Environment" (May 31 – June 2, 1999, Berlin).

Frutschi, H.-U. (1991), Erhöhtes Heizpotential von Gas und Öl beim Einsatz thermodynamischer Heizmethoden, ABB Technik 4/91, 13-20.

Gilli, P.V., Nakicenovic, N., Kurz, R. (1995) First and Second-Law Efficiencies of the Global and Regional Energy Systems, 16th Congress of the World Energy Council (Tokyo, 1995).

Gilli, P.V., Halozan, H., Streicher, W. (1992), Impact of Heat Pumps on the Greenhouse Effekt. HPC-AR1, IEA Heat Pump Centre, Sittard, the Netherlands.

Gilli, P.V., Streicher, W., Halozan, H., Breembroek, G. (1999), Environmental benefits of heat pumping Technologies. HPC-AR6, IEA Pump Centre, Sittard, The Netherlands.

Halozan, H., Kruse, H., Pettersen, J. (1996) European Heat Pump Research with Advanved Refrigerants. 5'th IEA Heat Pump Conference, Toronto, Canada.

Halozan, H. (1995) Propane for Heat Pumps, 19th Congress IIR/IIF (20 - 26 August 1995, the Netherlands).

Halozan, H. (1994), Propane - A Realistic Alternative, International Conference of New Applications of Natural Working Fluids in Refrigeration and Air Conditioning (May 10 - 13, Hannover), 331-338.

Halozan, H. (1994) Montreal, London, Copenhagen - What are the Results ?, IEA/HPC Workshop, Göteborg.

Halozan, H., Ebner, T (1994), Testing the R-12 and R-22 Alternatives R-134a, R-152a und R-290, IEA HPC Newsletter, 1/94, 32-34.

Halozan, H. (1990), Environmentally Acceptable Refrigerants - Effects on Heat Pump Development. 3rd International Workshop on Research Activities on Advanced Heat Pumps (Graz, Austria, 26-27 September 1990), dbv-Verlag, Graz, 198-204.

Halozan, H., Rieberer, R. (1999): 'CO₂ – eine interessante Alternative', KI Luft- und Kältetechnik 1/1999, 20 –23.

Halozan, H., Rieberer, R. (1999): Heat Pumps in Low-Heating-Energy Buildings, 20th International Congress of Refrigeration, IIR/IIF (Sydney, 1999).

Hecker, T., Müller, H. (1999), The next generation of heat pumps. 6th IEA Heat Pump Conference "Heat Pumps – a Benefit for the Environment" (May 31 – June 2, 1999, Berlin).

IEA HPC (1997), Heat Pumps can cut global emissions by more than 6 % - Renewable energy for a cleaner future. IEA Heat Pump Centre, Sittard, the Netherlands.

IEA HPC (1997a), Industrial Heat Pumps - Recent Experiences, Environmental and Production Benefits (Final Report of Annex 21). Report No. HPP-AN21-1. IEA Heat Pump Centre, Sittard, the Netherlands.

IEA HPC (1997b), Industrial Heat Pumps - Recent Experiences, Environmental and Production Benefits (Workshop Proceedings Windsor, Ontario, October 1997). Report No. HPP-AN21-4. IEA Heat Pump Centre, Sittard, the Netherlands.

Itoh, H. (18998), The Worlds Best Selling Heat Pump. Newsletter IEA HPC, Vol. 13, No. 3, 31-34.

Kodama, K. (1999), Market Situation for Gas Driven Heat Pumps. Plenary Presentation 8, 6th IEA Heat Pump Conference "Heat Pumps – a Benefit for the Environment" (May 31 – June 2, 1999, Berlin).

Laue, H.-J. (1999), Regional Report: "Heat Pump – Status and Trends" Europe. Plenary Presentation 3, 6th IEA Heat Pump Conference "Heat Pumps – a Benefit for the Environment" (May 31 – June 2, 1999, Berlin).

Lenarduzzi, F.J. (1999), Summary of Annex 23: Heat Pumps For Single-Room Applications. Poster Presentation 28, 6th IEA Heat Pump Conference "Heat Pumps – a Benefit for the Environment" (May 31 – June 2, 1999, Berlin).

Lorentzen, G. (1994), Use of CO₂ in Commercial Refrigeration - an Energy Efficient Solution, Conf. "New Applications of Natural Working Fluids in Refrigeration and Air Conditioning" (Hannover, 10 -13 May 1994).

Lorentzen, G. (1995): The Use of Natural Refrigerants: a Complete Solution to the CFC/HCFC Predicament, Int. J. Refrig. Vol. 18, No 3, 190 - 197.

Nishimura, T. (1999), Regional Reports: "Heat Pumps – Status and Trends" in Asia and Pacific. Plenary Presentation 2, 6th IEA Heat Pump Conference "Heat Pumps – a Benefit for the Environment" (May 31 – June 2, 1999, Berlin).

Pettersen J., Lorentzen G. (1993), Eine neue, effiziente und umweltfreundliche PKW-Klimaanlage mit CO₂ als Kältemittel. Luft- und Kältetechnik 3/1993, 105.

Rieberer, R., Kasper, G., Halozan, H. (1997), CO_2 - A Chance for Once-Through Heat Pump Water Heaters. IEA/IIR Workshop: "CO₂ Technologies in Refrigeration, Heat pump and Air Conditioning Systems" (Trondheim, Norway).

Rieberer, R. (1998): CO₂ as Working Fluid for Heat Pumps, Ph.D. thesis, Institute of Thermal Engineering, Graz University of Technology.

Rieberer, R., Halozan, H. (1998): CO₂ Heat Pump Water Heater: Simulation and Test Results, International Refrigeration Conference at Purdue University, West Lafayette, USA, 133 - 138.

Rieberer, R., Halozan, H. (1998), CO₂ Heat Pumps in Controlled Ventilation Systems, IIR Conference Natural Working Fluids, (Oslo, Norway, June 2-5, 1998).

Schmidt, E.L., Klöcker, K., Flacke, N., Steimle, F. (1998), Applying the transcritical CO₂ process to a drying heat pump. International Journal of Refrigeration, 3/21, 194-201.

Stuij, B., Stene, J. (1994) International Heat Pump Status and Policy Review, Part 1 – Analysis. Analysis Report No. HPC-AR3, IEA Heat Pump Centre Sittard, the Netherlands/IIR.

Stuij, B. (1995) Residential Heat Pump Applications – An International Overview, Newsletter, IEA Heat Pump Centre, Sittard, the Netherlands, Vol. 13, No. 3, 12-21.

WEC (1988), Environmental Effects Arising from Electricity Supply and Utilization and the Resulting Costs to the Utility. Report 1988, WEC, London.

18TH WORLD ENERGY CONGRESS

HEAT PUMPS FOR DIFFERENT WORLD REGIONS – NOW AND IN THE FUTURE

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SUMMARY

Heat pumps offer the possibility of reducing energy consumption significantly, mainly in the building sector, but also in industry. The drive energy is most commonly electricity. Ground-source heat pumps combined with low-temperature heat distribution systems achieve seasonal performance factors (SPFs) of 4 and higher. The second law thermodynamics shows the advantages: while a condensing boiler can reach a primary energy ratio (PER) of 105 % (the theoretical maximum would be 110 % based on the lower calorific value), heat pumps achieve 200 % and more, based on hydro or wind energy 400% or more. Further improvements of power plant efficiency and heat pump SPF are possible in the future.

The majority of heat pumps is in operation in Japan, in China and in the US, most commonly air-to-air dual-mode units for both heating and cooling, China has already the largest industry for producing air conditioners, and in South East Asia the trend to this technology is rising rapidly. Europe has been concentrating on heating-only hydronic heat pumps and heat recovery systems, but sales of dual-mode units are growing.

Presently more than 100 million heat pumps with a thermal output of about 1300 TWh/a are in operation world-wide, reducing CO_2 emissions by 0.13 Gt/a. The potential for reducing CO_2 emissions assuming a 30 % share in the building sector using technology presently available is about 6 % of the total world-wide CO_2 emission of 22 Gt/a. Therefore, heat pumps are one of the key technologies available on the market for energy conservation and reducing CO_2 emission.

18 IEME CONGRES MONDIAL DE L`ENERGIE

POMPES DE CHALEUR POUR DIFFERENTES PARTIES DU MONDE – AUJOURD'HUI ET A L'AVENIR

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RESUMÉ

Les pompes à chaleur offrent la possibilité de réduire notablement la consommation d'énergie, principal dans le secteur de bâtiment, mais aussi dans l'industrie. La puissance de propulsion est dans l'électricité généralement. Les systemes pour la distribution de chaleur avec la température inférieure combine des pompes à chaleur de terre, les coefficients performance annuelles SPFs atteignent de 4 et sur cela. La deuxième loi de la thermodynamique montre les avantages: tandis qu'une chaudiere de la condensation marmite une liaison d'énergie primaire, PAR, de 105% peut atteindre, le maximum théorique serait 110%, référentiel sur le pouvoir calorifique inférieur, des pompes à chaleur en atteignent 200% et plus, référentiel sur force hydraulique ou énergie de vent 400% ou plus. Autres améliorations de rendements de centrale électrique et les coefficientes performance saisonales de pompes à chaleur dans l'avenir sont possible.

La majorité des pompes à chaleur est au Japon, en Chine et dans les U.S.A. dans l'exploitation, dans l air/air`pompes à chaleur des installations de conditionnement d`air réversible et général pour le chauffage aussi comme pour la réfrigération. La Chine a l'industrie la plus grande pour la production des installations de conditionnement d`air, et en Asie de sud-est, les nombres de vente augmentent rapide. L'Europe s'est concentrée sur la pompe à chaleur de chauffage pure dans la liaison avec des systems d`eau chaude pour la distribution de chaleur, mais la diffusion du l`air/air pompes à chaleur installation de conditionnement d`air réversible augmente aussi ici.

Les pompes à chaleur sont fortuitement mondiaux actuel 1300 TWh/a plus que 100 millions avec une livraison de chaleur dans l'exploitation que les CO2 émissions réduisent à 0.13 Gt/a. Le potentiel pour une 30% quotité dans le secteur de bâtiment est à peu près 6% de la CO₂ émission mondiale de 22 Gt/a. Donc est des pompes à chaleur un des technologies de clé sur le marché pour économie d'énergie et la réduction de la CO₂ émission.